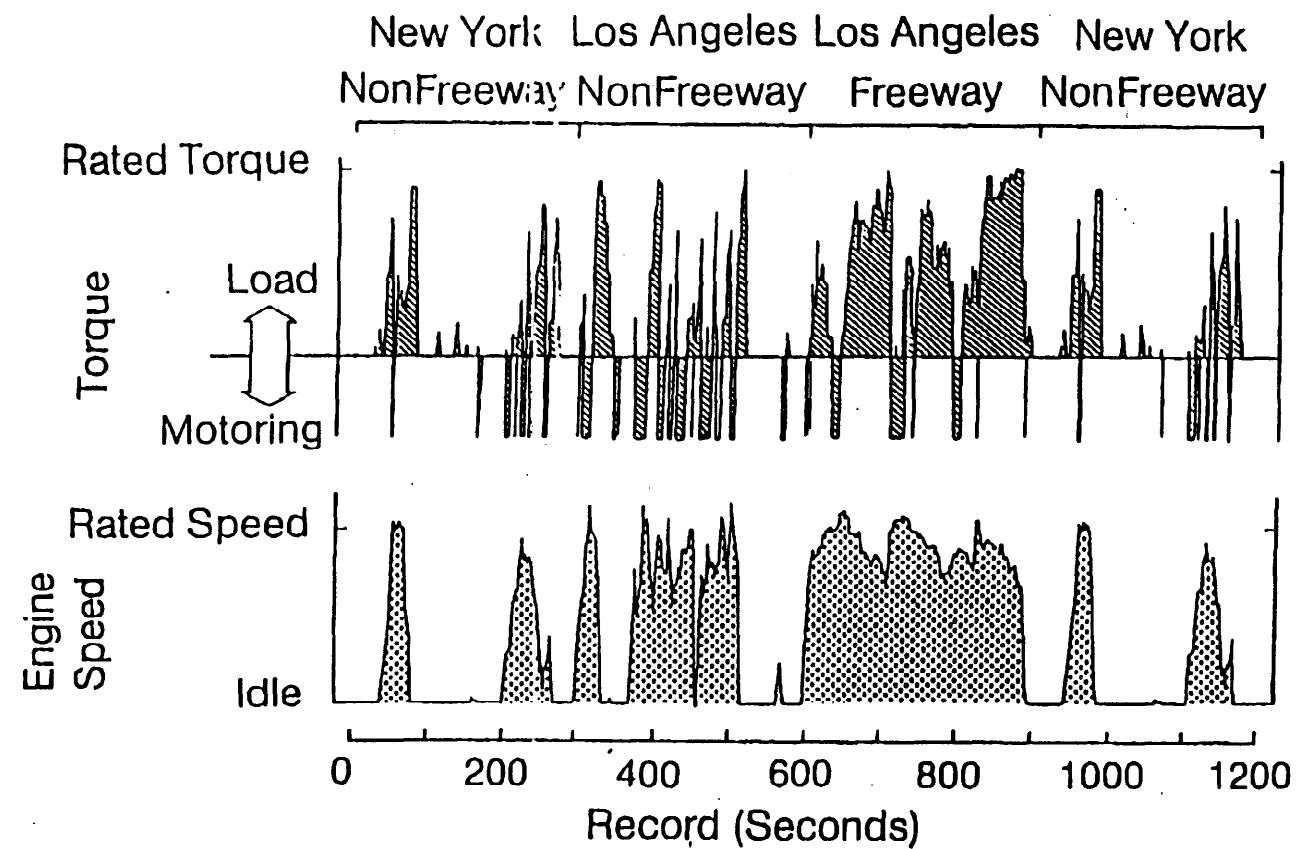
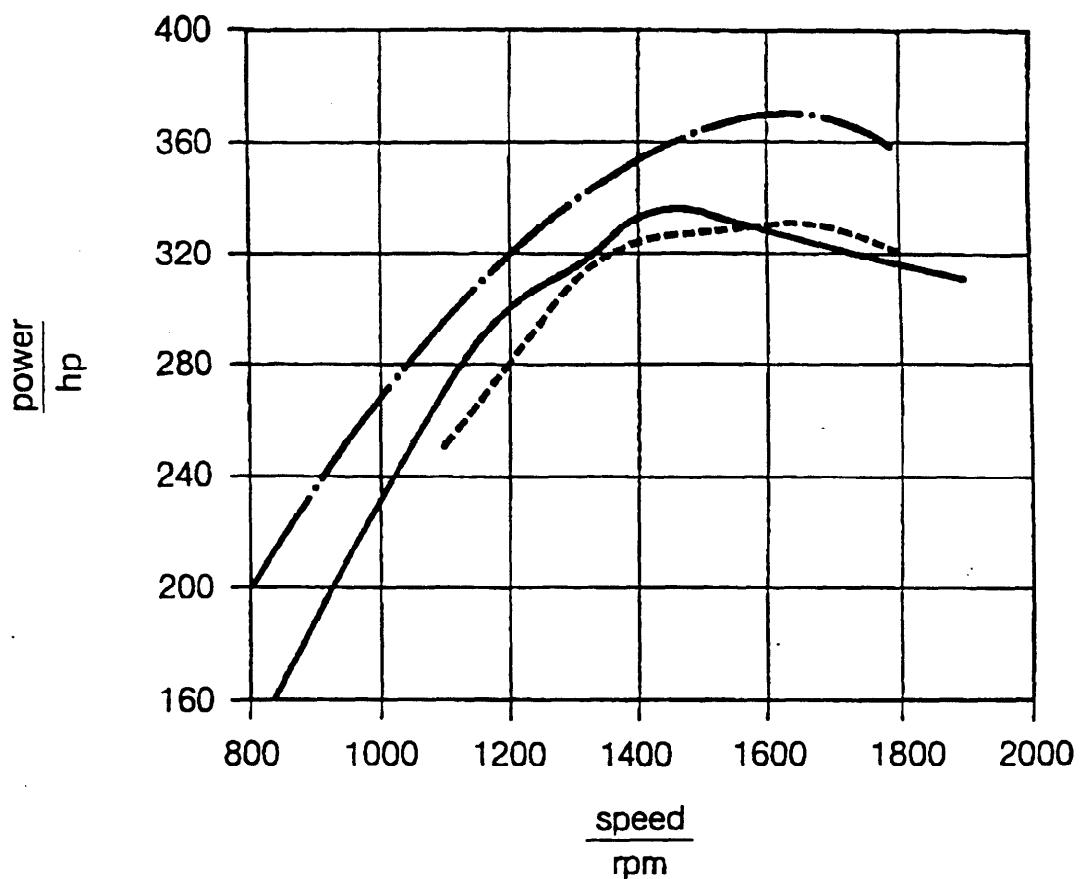
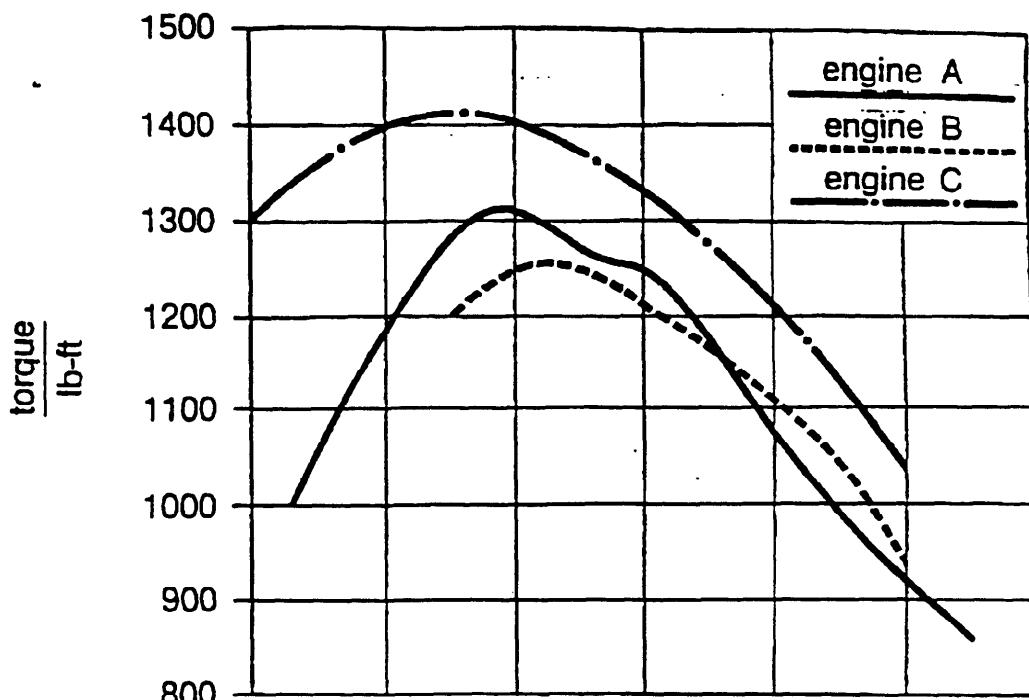


# EXHIBIT C



**Torque- and Speedflow  
USA-Transient-Test cycle for Heavy-Duty-Diesel-  
Engines**

Geschäftsbericht NFZ  
Entwicklung  
EN / MVT 8710-61

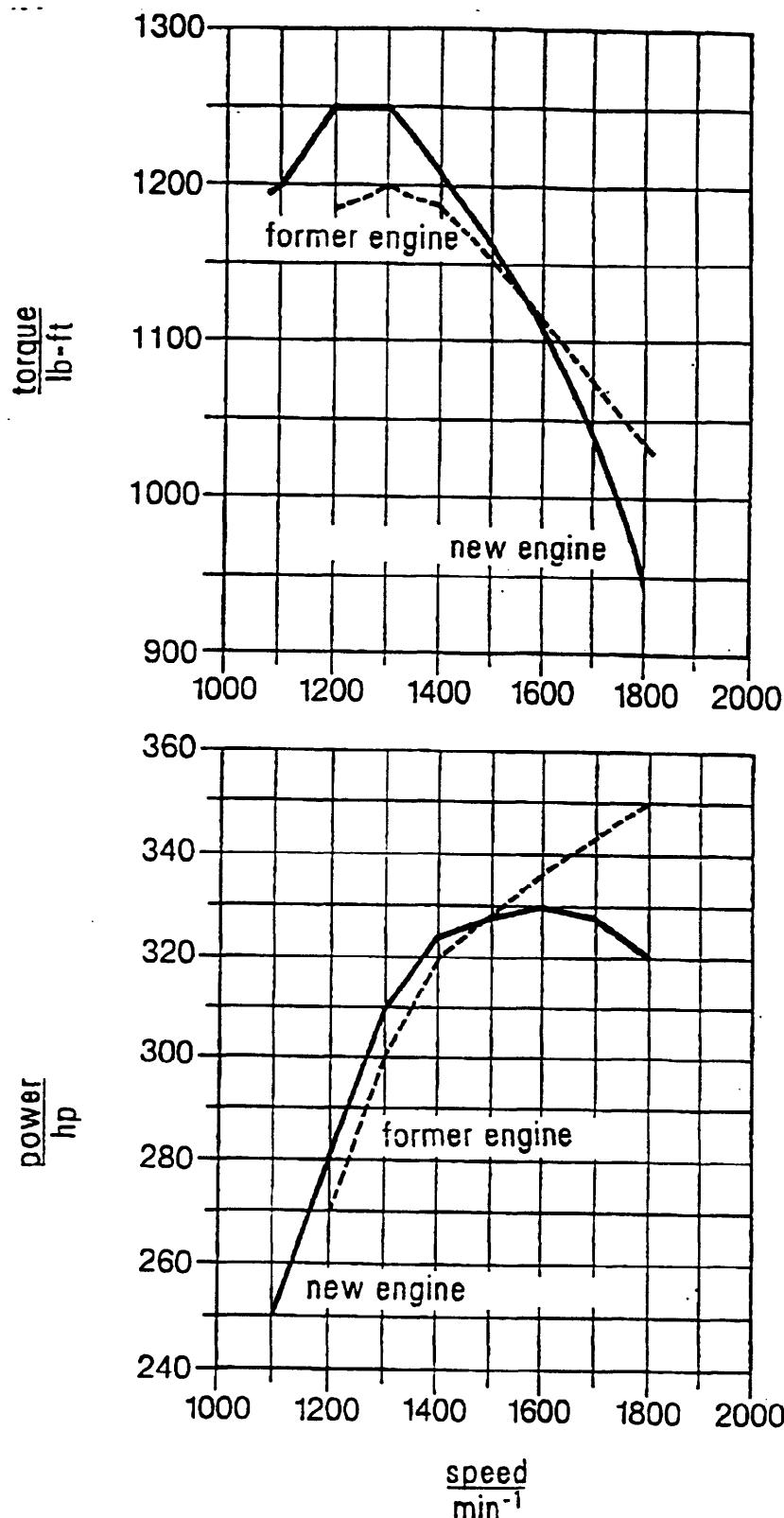


Mercedes-Benz  
Nutzfahrzeuge  
Entwicklung

B20 603 06 063 02 A

## Power and Torque Characteristics of modern US-Diesel-Engines

EN/MVT 9105-05



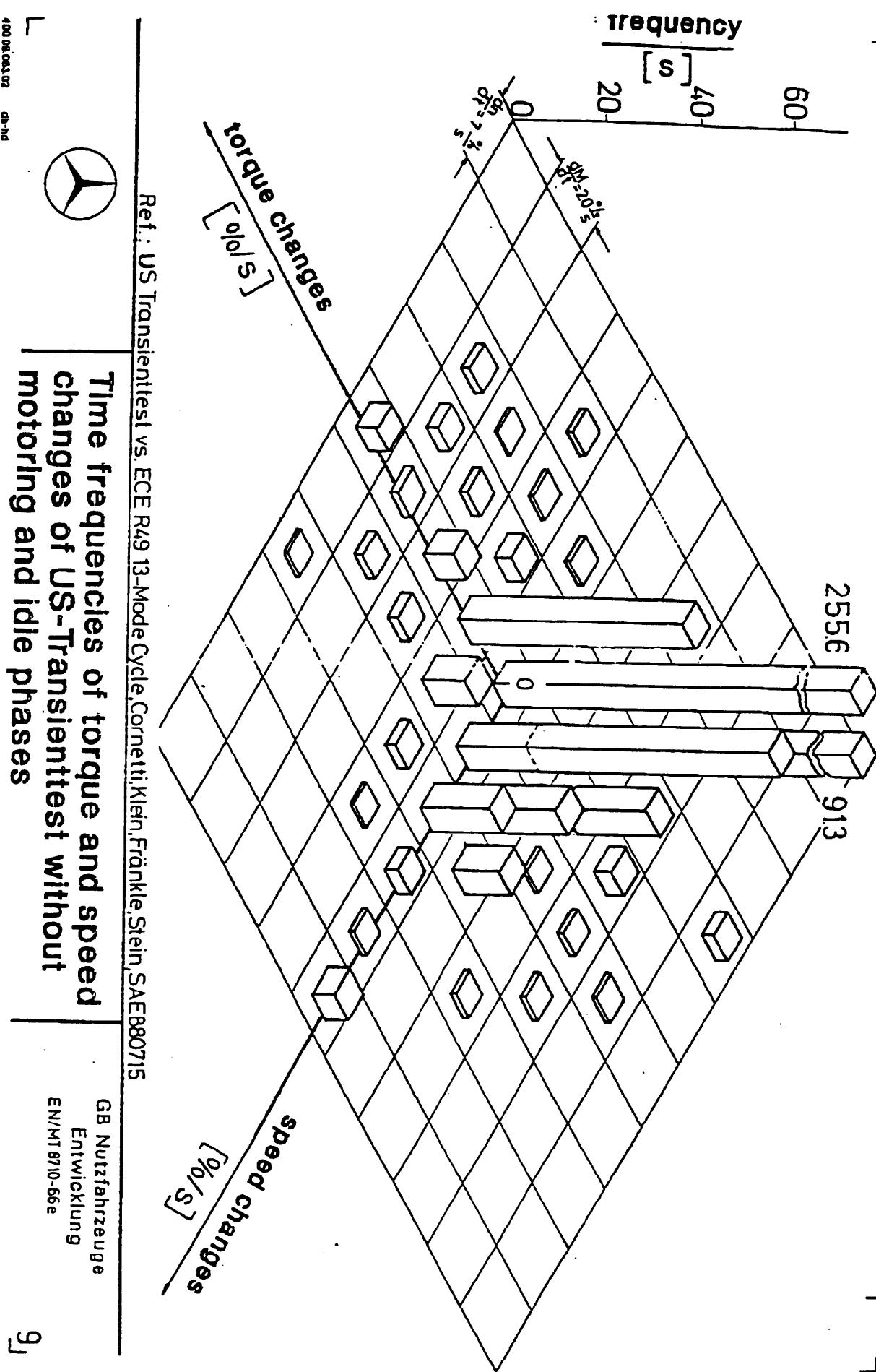
Mercedes-Benz  
Nutzfahrzeuge  
Entwicklung

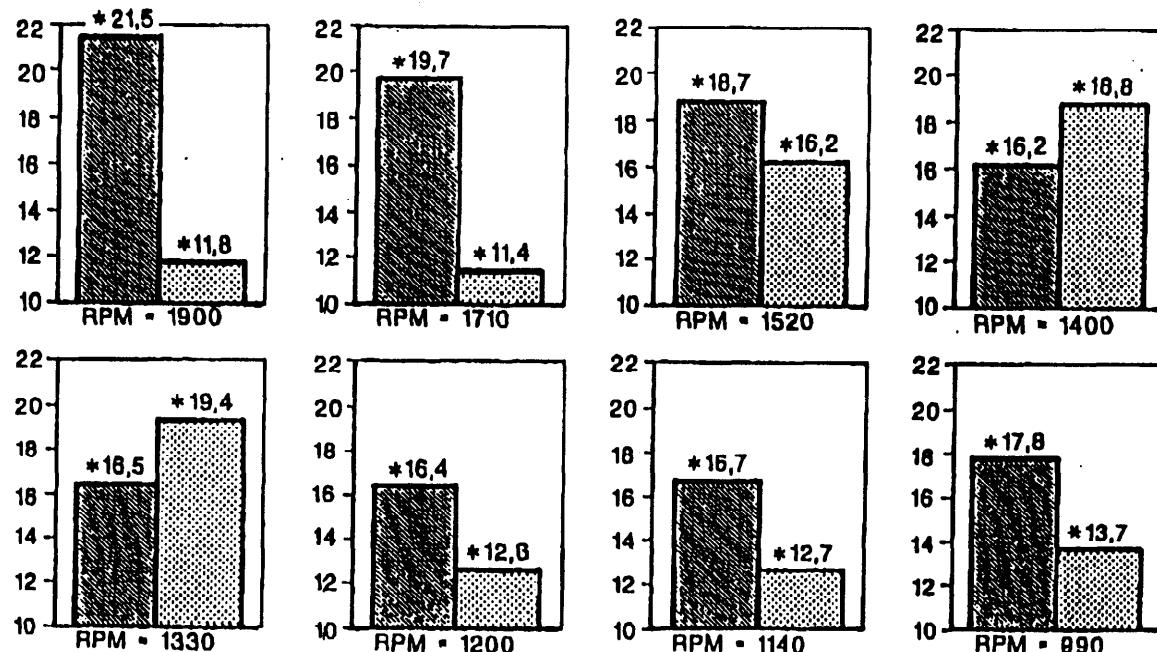
020.003.00.003.02 und Dmvt-fl

### Power and Torque Characteristics of modern US-Diesel-Engines Engine B

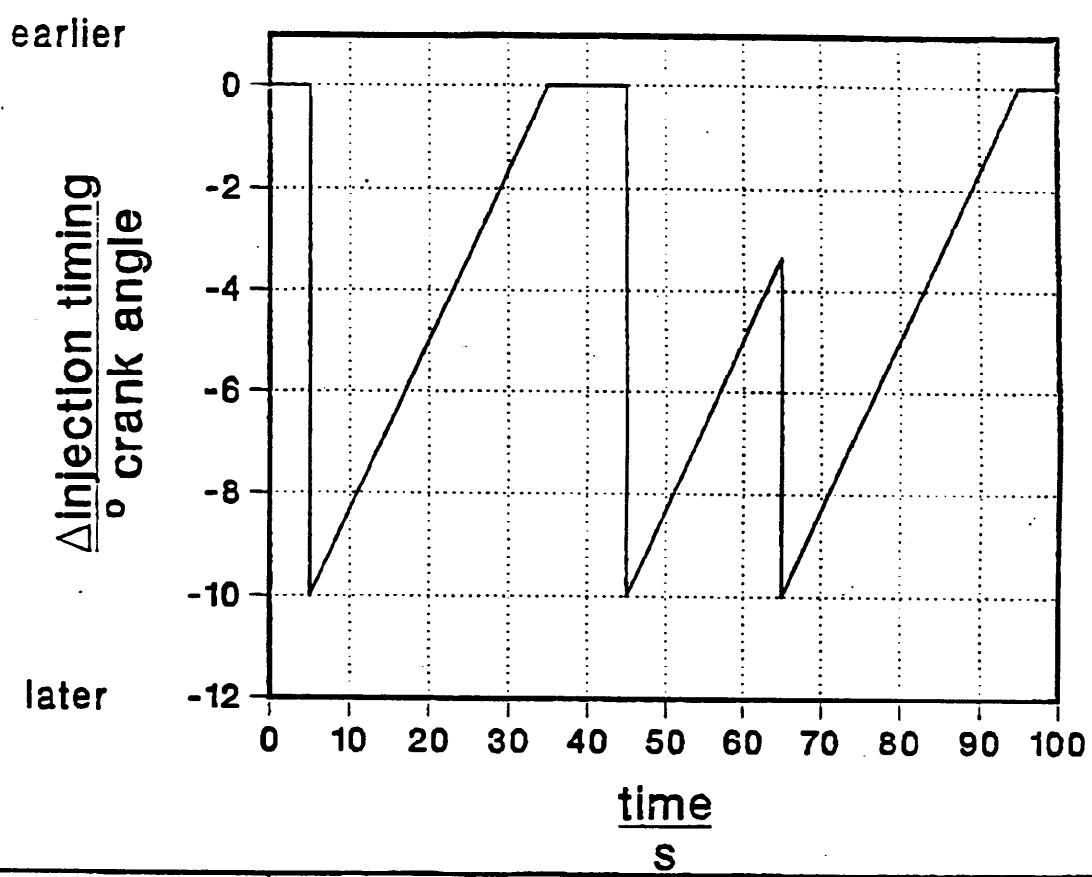
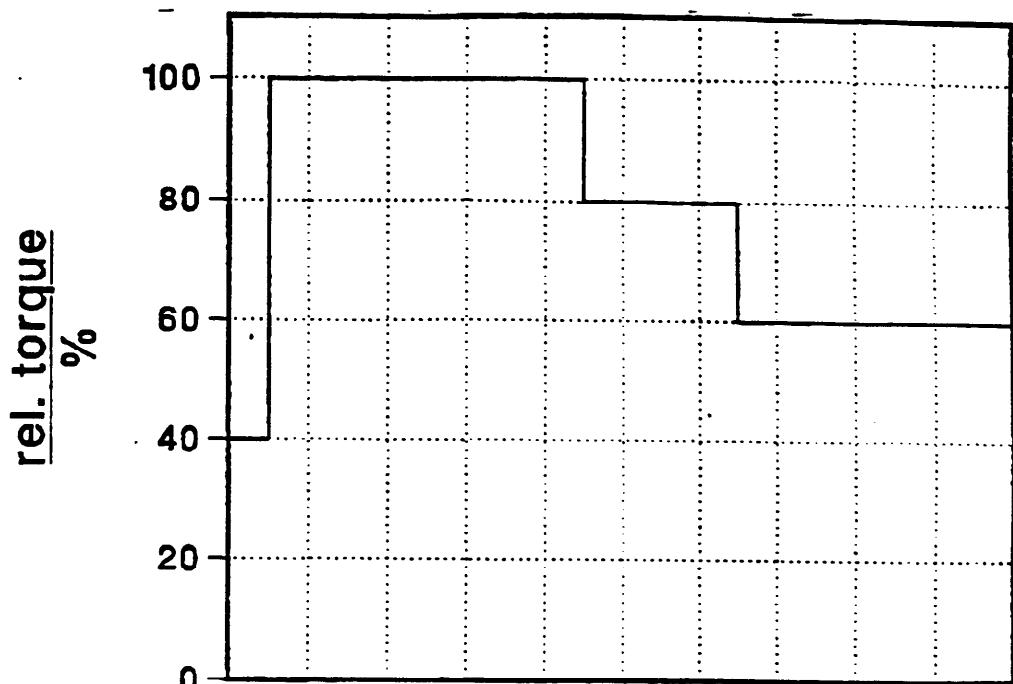
EN/MVT 9102-03

7





electr.injection start, steady electr.injection start, unsteady \*start of ignition °crank before TDC



Mercedes-Benz  
Nutzfahrzeuge  
Entwicklung

820.501.00.053.C2 AND DWV/T-1

schematic characteristic of injection  
timing at constant and transient load  
(several US engine manufacturers)

EN/MVT 9104-09

# **EXHIBIT D**

# **THE EFFECT OF ELECTRONIC CONTROLS ON THE TRANSIENT TEST**

**PREPARED BY:**

**Michael J. Samulski**

**August 14, 1991**

## **OUTLINE OF BRIEFING**

- **Definition of defeat device**
- **Mercedes-Benz presentation**
- **Present investigation of designing to the test cycle**
- **Representativeness of the transient test**
- **Further investigation of designing to the test cycle**
- **Conclusions**
- **Tabular emission and fuel consumption data for the certified 1991 HHDD engines**

## **DEFEAT DEVICE**

- "Defeat Device" means an auxiliary emission control device (AECD) that reduces the effectiveness of the emission control system under conditions which may reasonably be expected to be encountered in normal vehicle operation and use, unless
  - (1) such conditions are substantially included in the Federal emission test procedure,
  - (2) the need for the AECD is justified in terms of protecting the vehicle against damage or accident, or
  - (3) the AECD does not go beyond the requirements of engine starting.
- Any incongruous strategy which is triggered by sensing a change in operating conditions may be a defeat device.

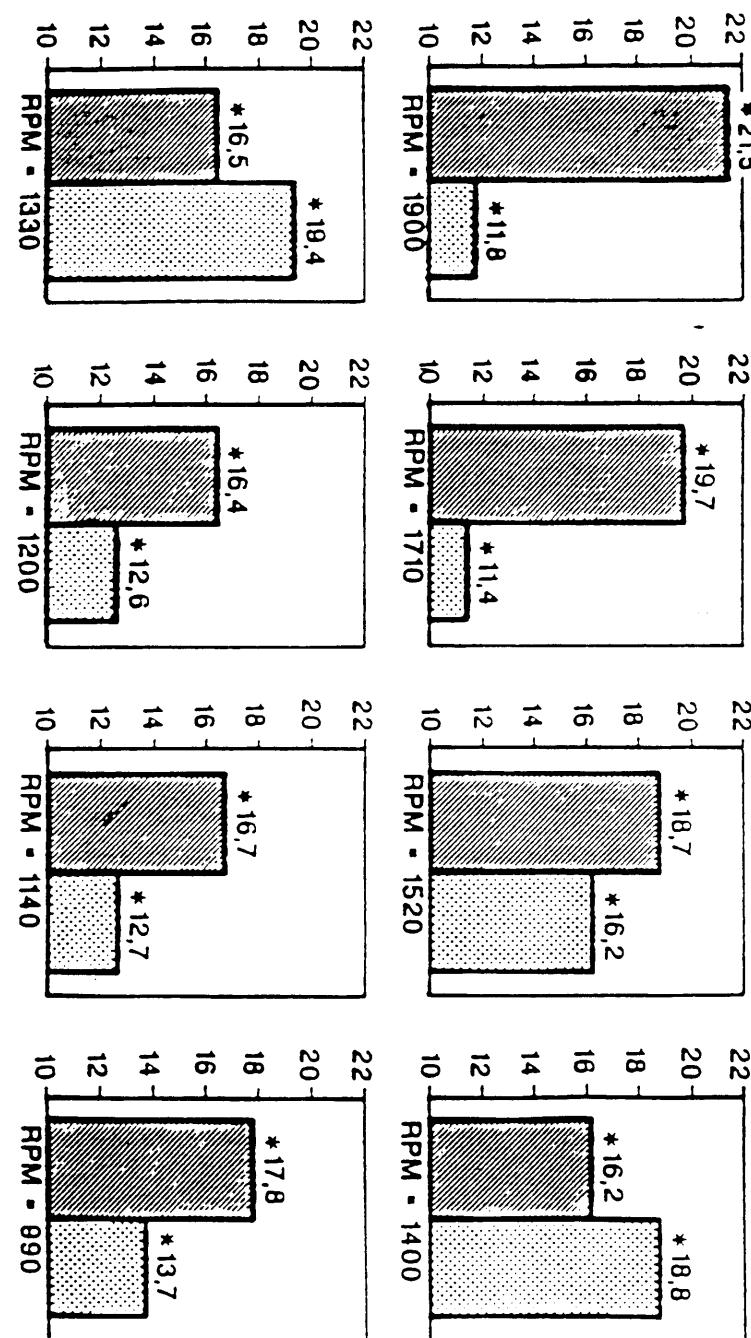
## MERCEDES-BENZ PRESENTATION

- The claim was made that U.S. manufacturers are designing their engines strictly to perform well on the transient test by using electronic control to retard the timing during transience
  - When the timing is retarded due to transience, it takes 30 seconds to return to normal
  - Proven with data from a 1991 HHDD engine
  - We suspect Detroit Diesel's Series-60 engines
- The claim was made that the transient cycle is not representative of actual driving
  - Claim: Drivers tend to operate a vehicle at the engine speed (RPM) where the torque is maximized
  - Claim: HHDD engines now reach their peak torques at lower RPM's

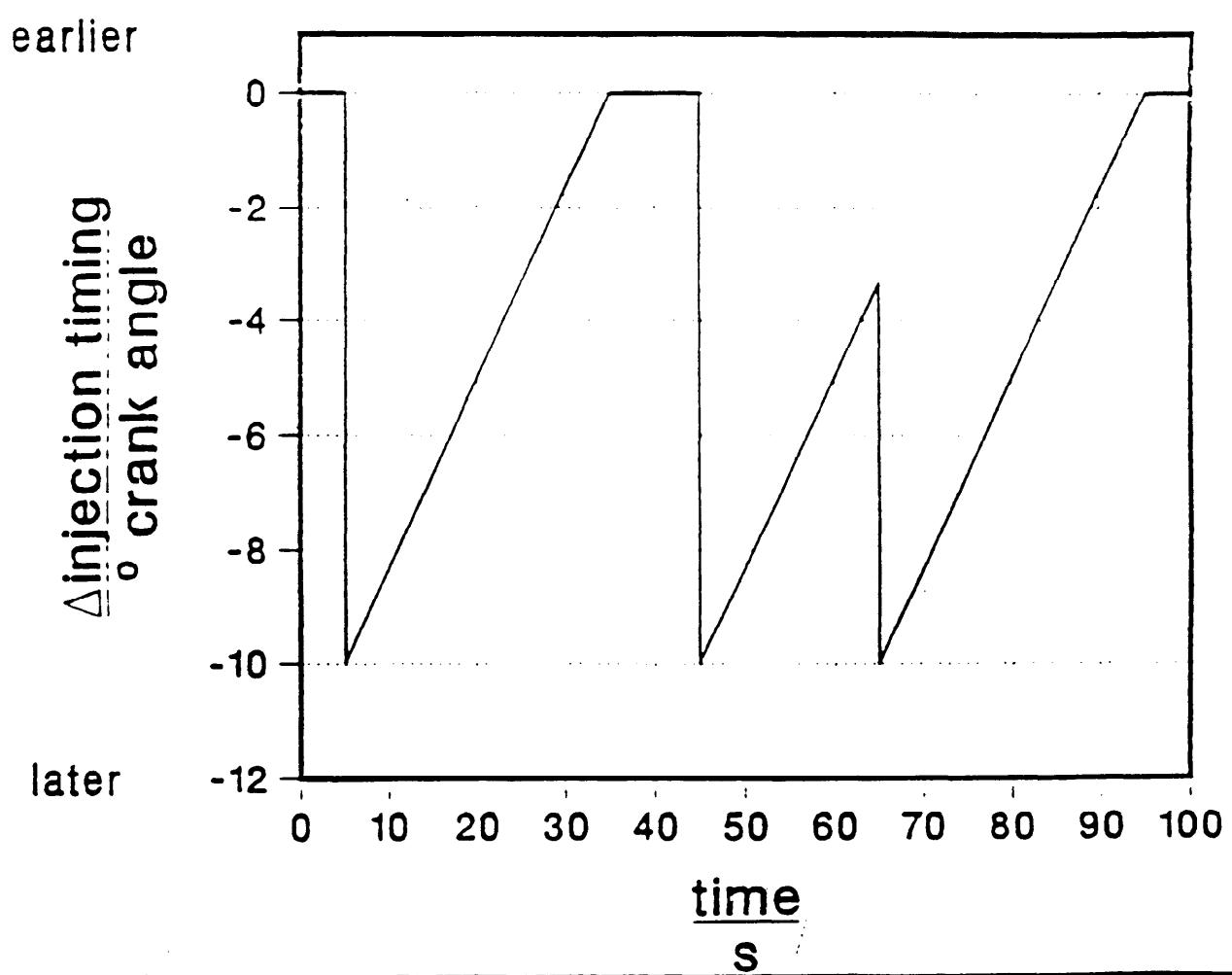
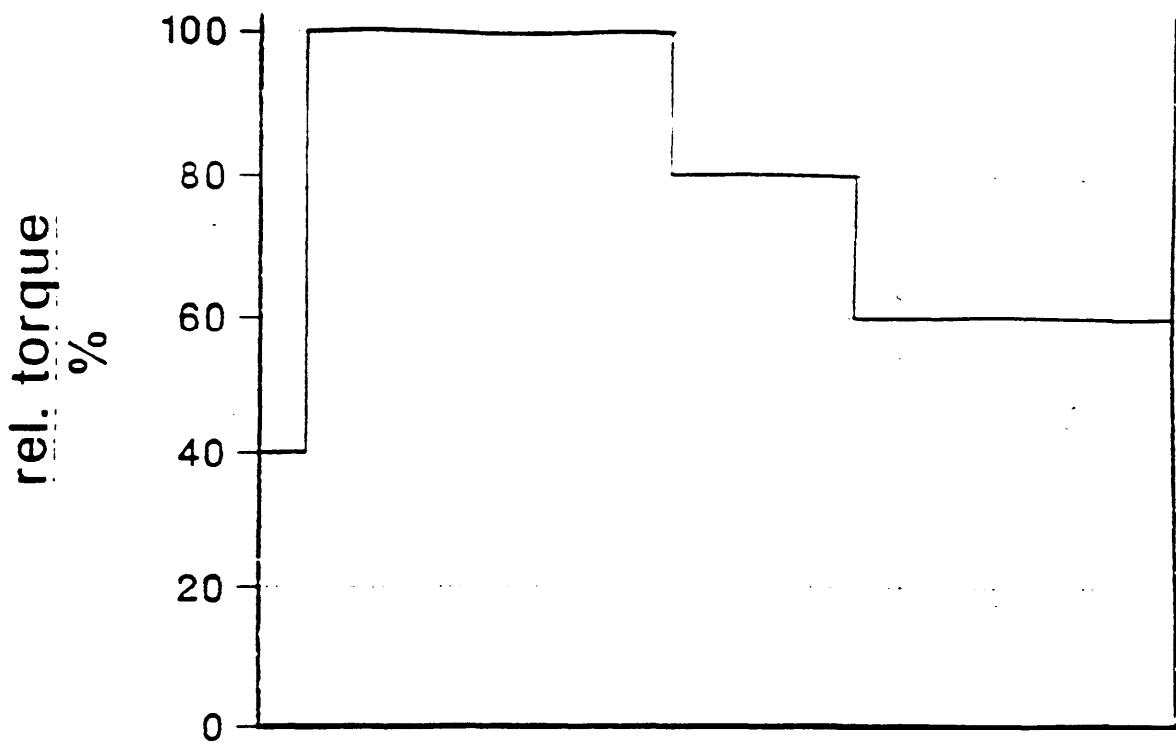


Mercedes-Benz  
Nutzfahrzeuge  
Entwicklung

Start of electronic injection at steady  
and transient full load of modern  
US-Diesel engines, engine A



■ electr.injection start, steady ■ electr. injection start, unsteady \*start of ignition • crank before TDC



Mercedes-Benz  
Nutzfahrzeuge  
Entwicklung

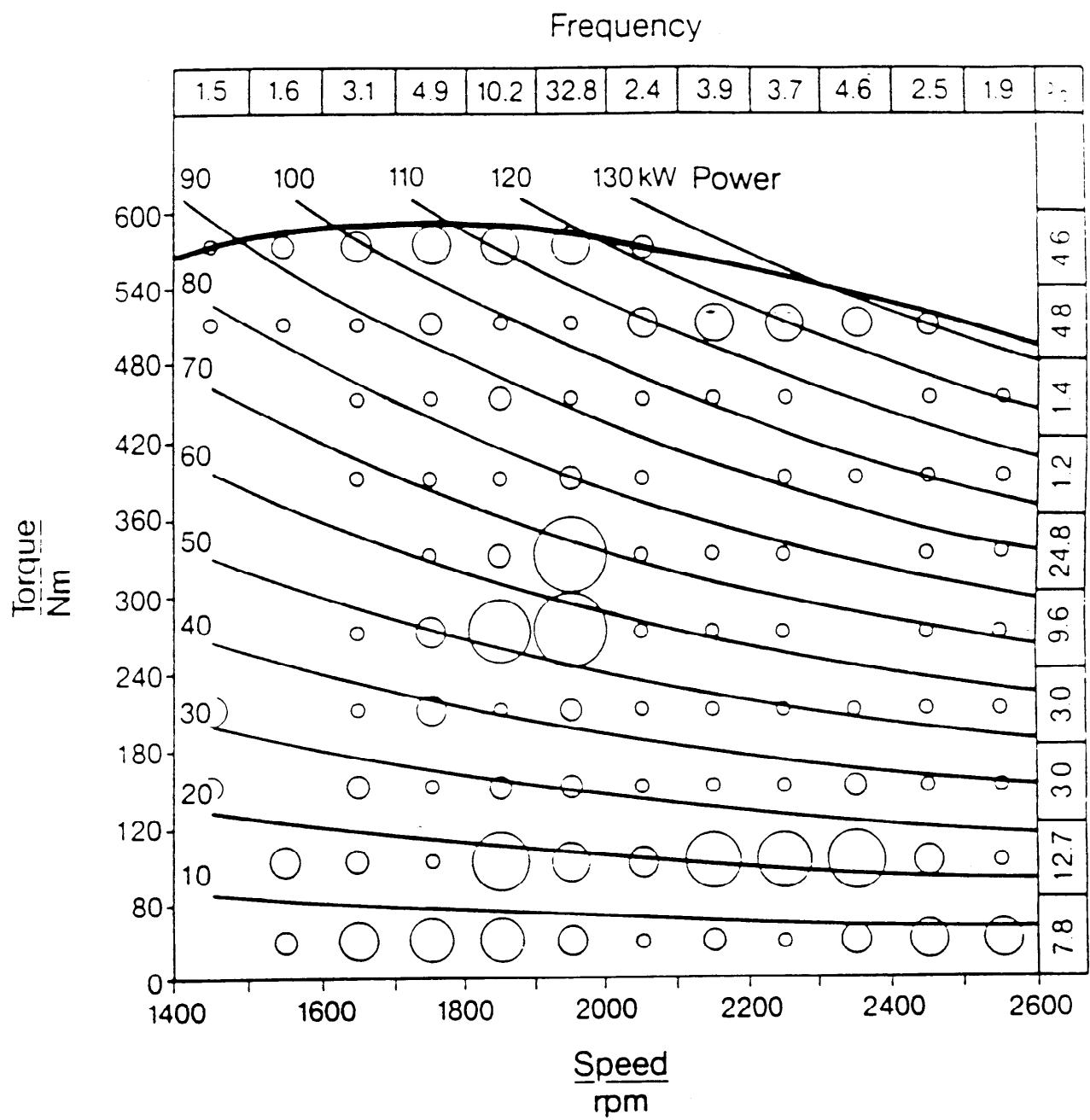
820.603.00.063.02 DURFT-FL

schematic characteristic of injection  
timing at constant and transient load  
(several US engine manufacturers)

EN/MVT 9104-09

17

447



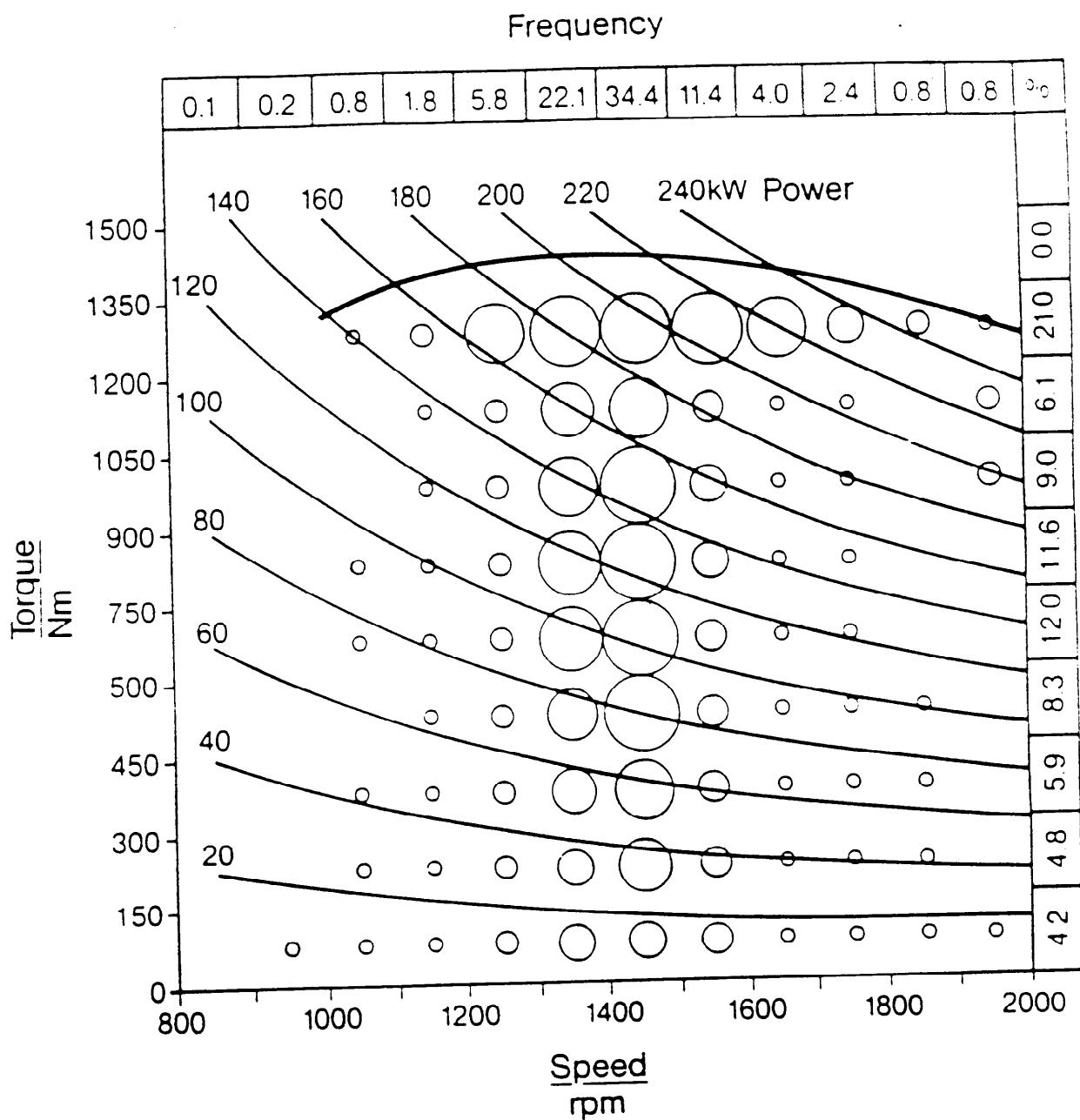
Lit.: Netherlands Delegation to MVEG



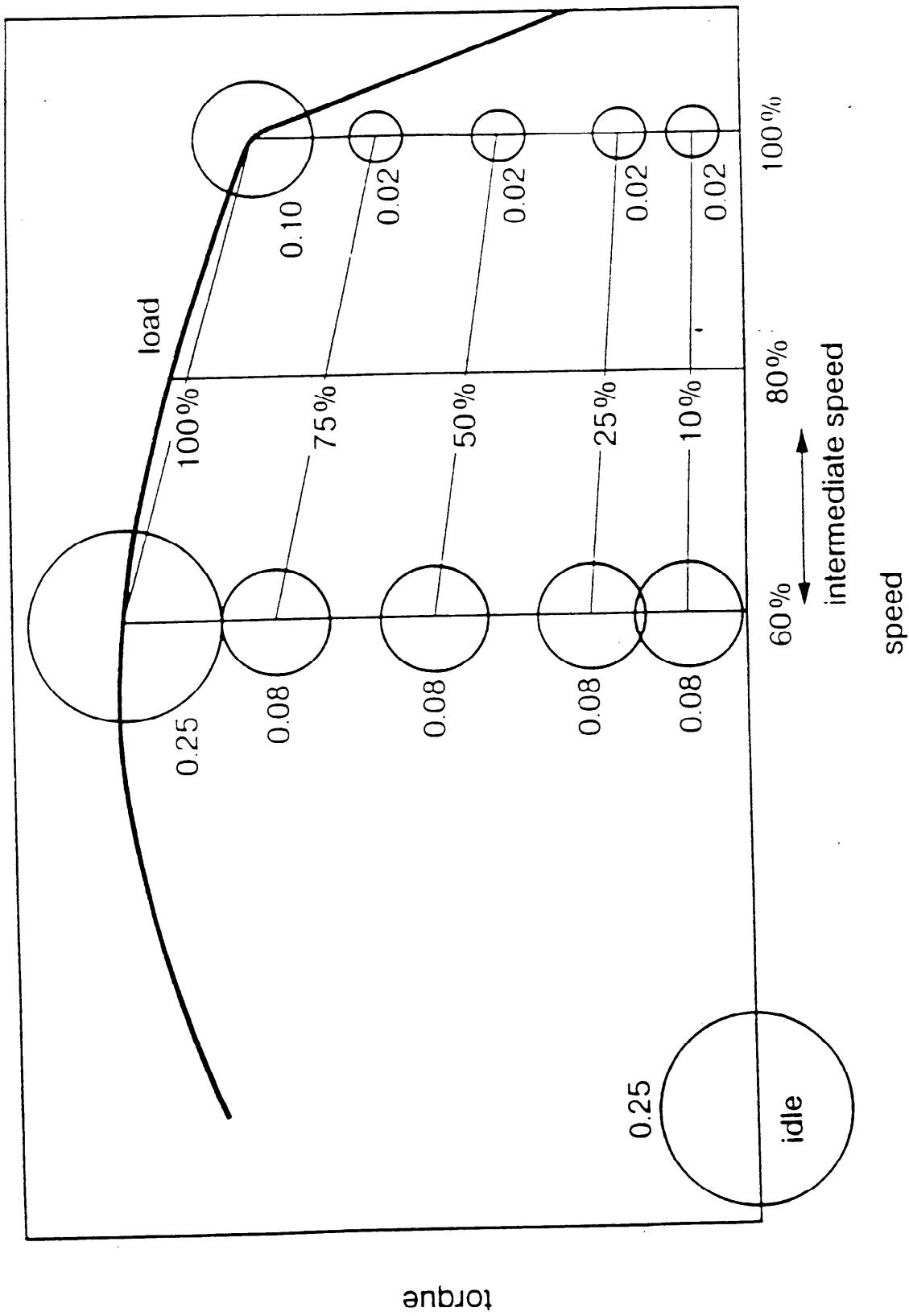
Mercedes-Benz

**Engine map frequencies of  
a 10t truck for urban traffic**

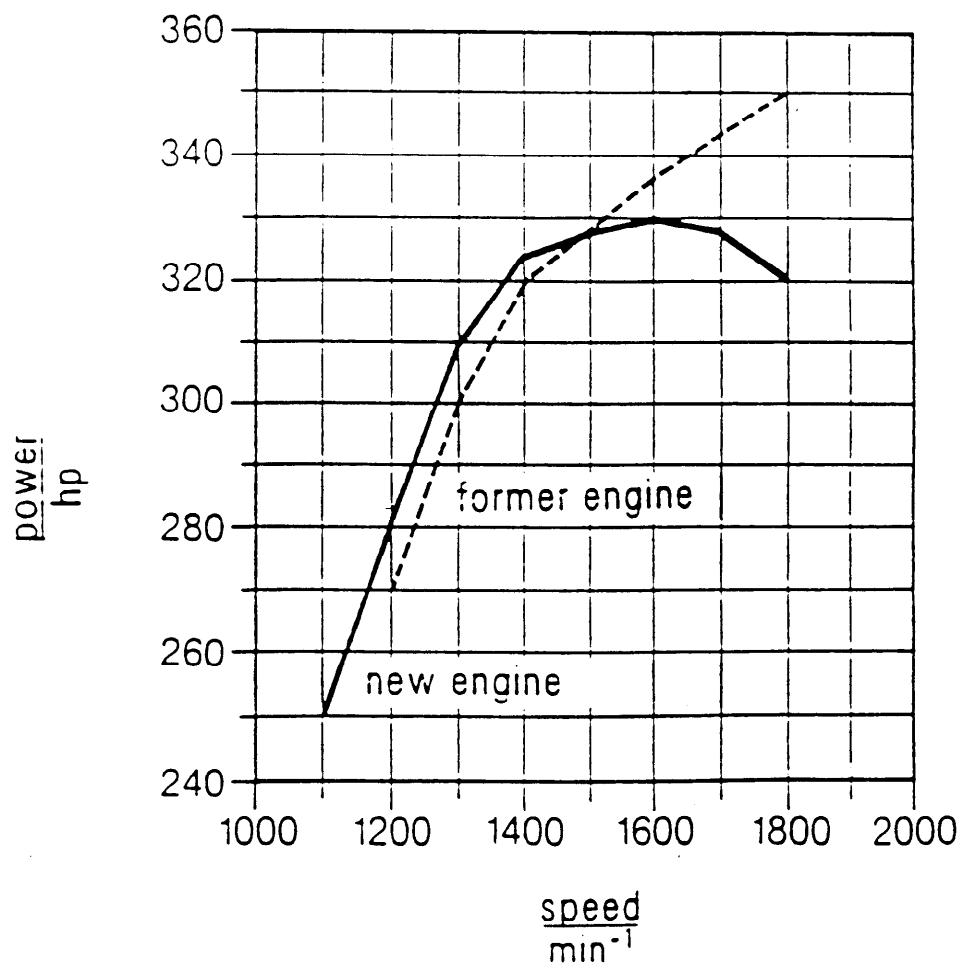
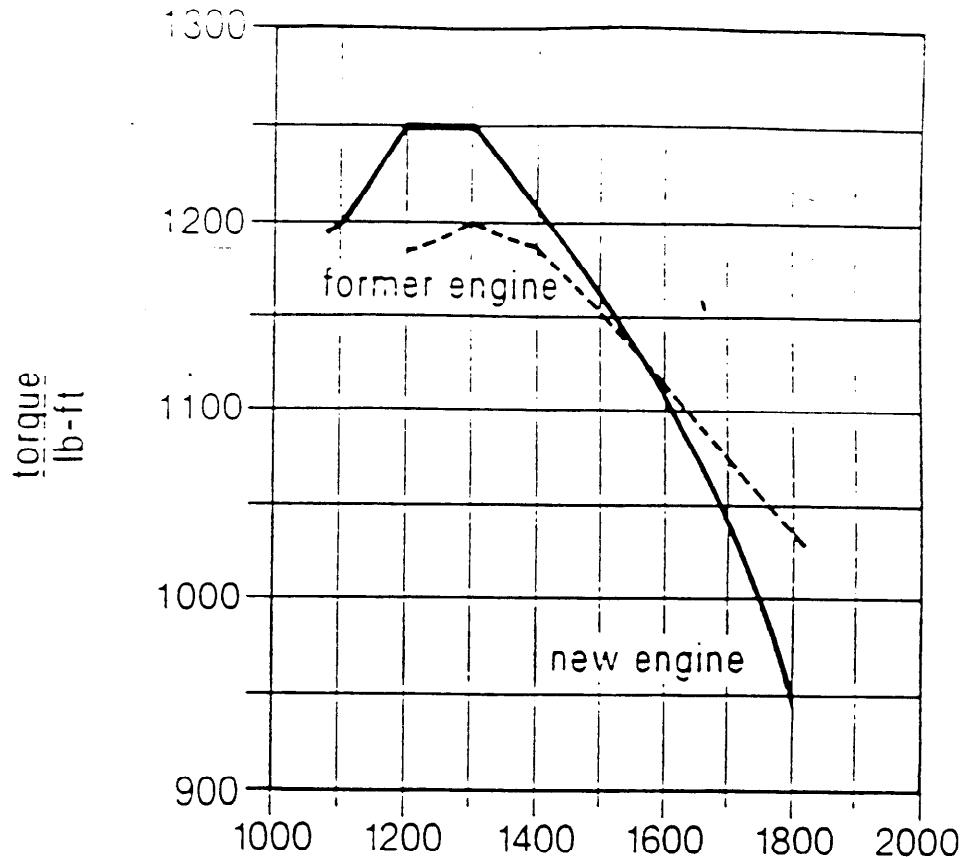
EN/MVT 9102



Lit.: Netherlands Delegation to MVEG



**Measuring points and weighting factors of  
ECE R49 13-mode test**



Mercedes-Benz  
Motorenfertigung

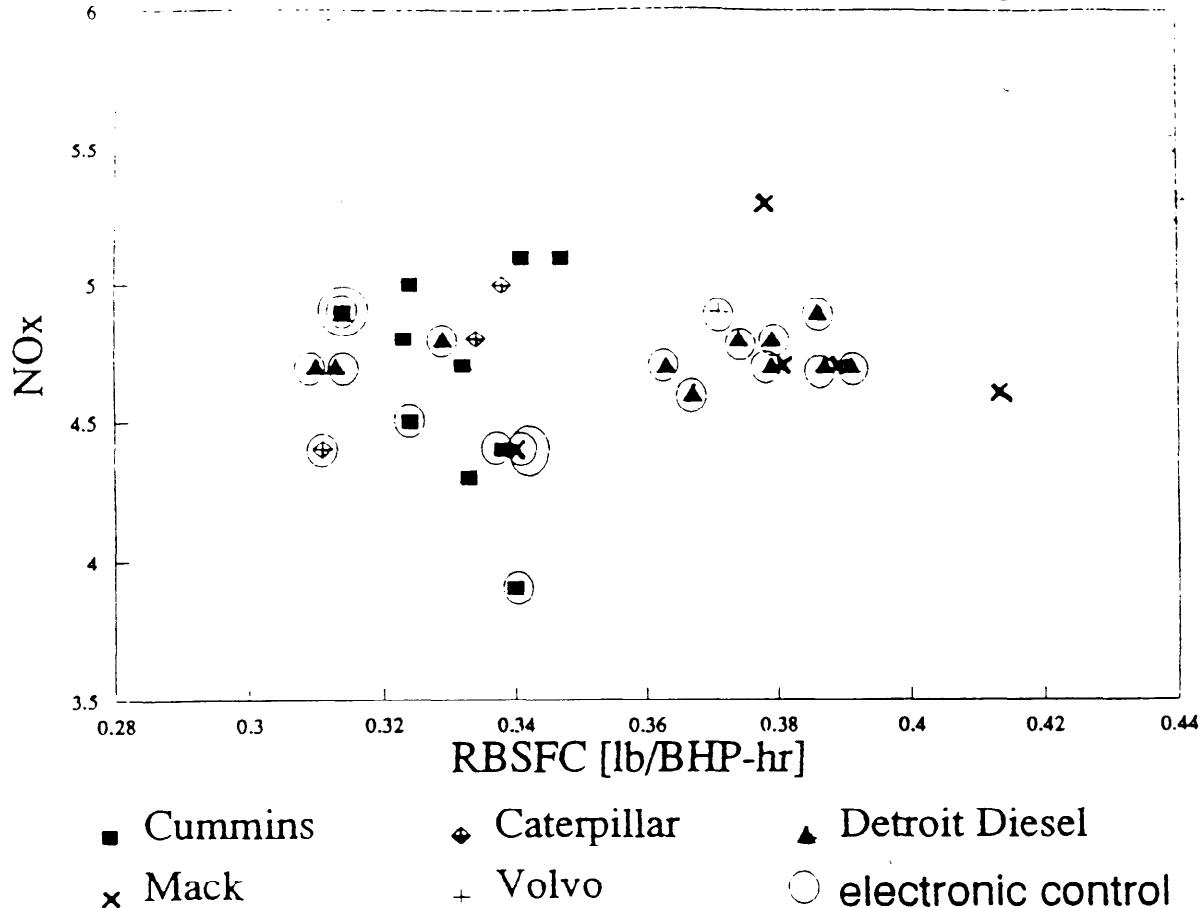
## Power and Torque Characteristics of modern US-Diesel-Engines

EN/MVT 9102-03

## **PRESENT INVESTIGATION**

- **A look at NOx versus fuel consumption**
  - **An attempt was made to discover a 1991 HHDD engine family with exceptionally low NOx**
  - **No such engine family was found**
- **A look at rated brake specific fuel consumption versus cycle brake specific fuel consumption**
  - **An attempt was made to discover a 1991 HHDD engine family which had an exceptionally poor match between its RBSFC and its CBSFC**
  - **No such engine family was found**

## RATED FUEL CONSUMPTION VS. NO<sub>x</sub> for 1991 certified heavy heavy-duty diesel engines

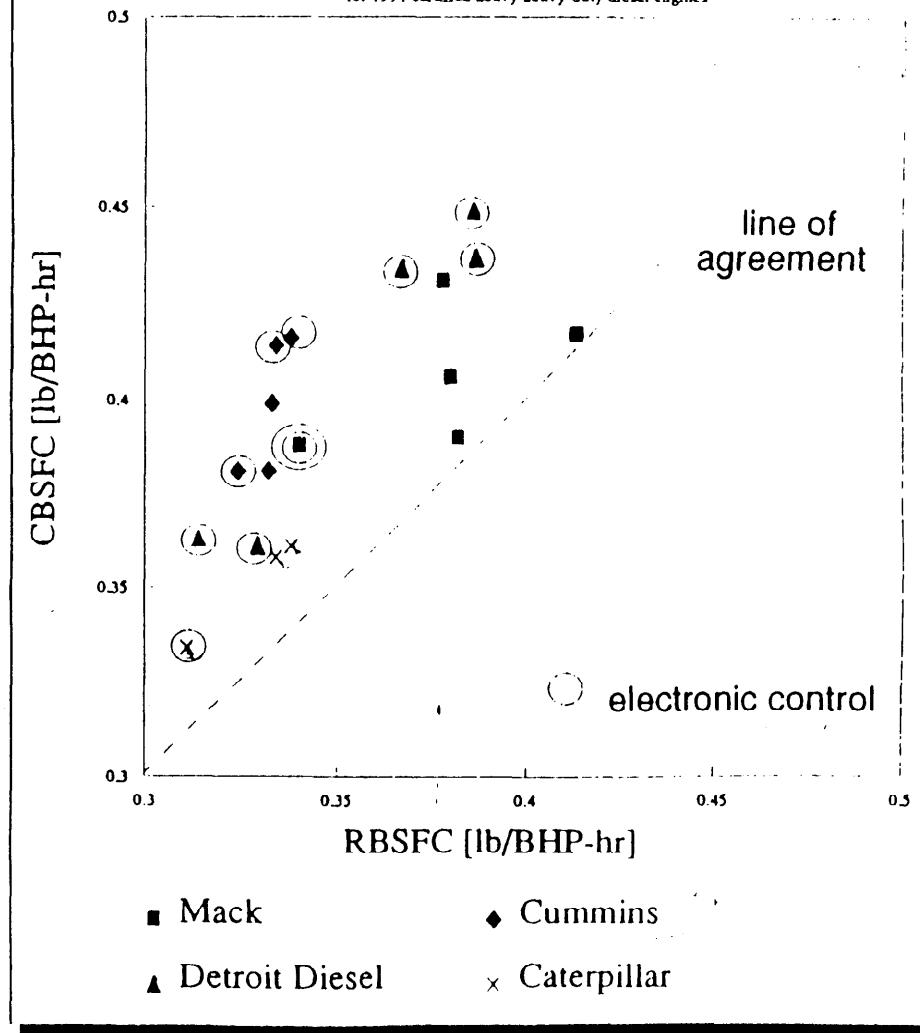


	<u>Mack</u>						
RBSFC	0.38	0.413	0.34	0.382	0.38	0.34	0.378
CBSFC	0.406	0.417	0.388	0.39	0.406	0.388	0.431
<u>Cummins</u>							
RBSFC      0.324    0.332    0.338    0.334    0.333							
CBSFC hot    0.378    0.379    0.413    0.41    0.396							
CBSFC cold    0.402    0.396    0.436    0.44    0.417							
CBSFC      0.381    0.381    0.416    0.414    0.399							
<u>Caterpillar</u>							
RBSFC      0.311    0.338    0.334							
CBSFC      0.334    0.361    0.358							
<u>Detroit Diesel</u>							
RBSFC      0.367    0.386    0.387    0.314    0.329							
CBSFC hot    0.427    0.444    0.434    0.362    0.359							
CBSFC cold    0.475    0.483    0.455    0.372    0.375							
CBSFC      0.434    0.449    0.437    0.363    0.361							

weightings are 1/7 cold and 6/7 hot

### TRANSIENT VS RATED FUEL CONSUMPTION

for 1991 certified heavy-duty diesel engines



## **REPRESENTATIVENESS OF THE TRANSIENT TEST**

- **Cycle distribution of the transient test is**
  - concentrated at idle and at 80% RPM
  - based on NY and LA on and off freeway study
- **Changes in torque characteristics since the transient test was created are not significant**
  - Heavy-duty diesel engines now reach their peak torque at lower RPM's than when the transient test was developed
  - However, the maximum rated RPM's have also been reduced
- **The only sections in the Heavy-Duty Diesel transient test cycle which achieve steady-state for more than ten seconds are at idle (zero torque)**

## ENGINE SPEED DENSITY GRAPH for Diesel Engine Transient Cycle

50

40

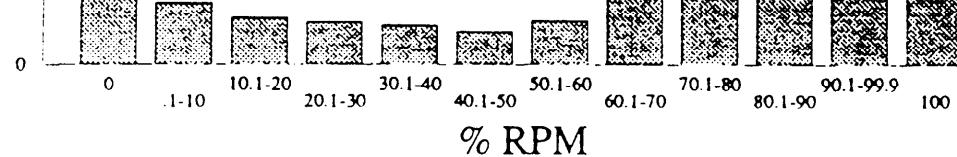
30

20

10

0

36% of density is  
zero load idling



% RPM



CUMMINS ENGINE COMPANY INC

Columbus Indiana 47201

BASIC ENGINE MODEL

NTF-365

CURVE NUMBER

RC 3544-A

## AUTOMOTIVE PERFORMANCE CURVE

ENGINE FAMILY

EPL CODE

DATE

8/21/75

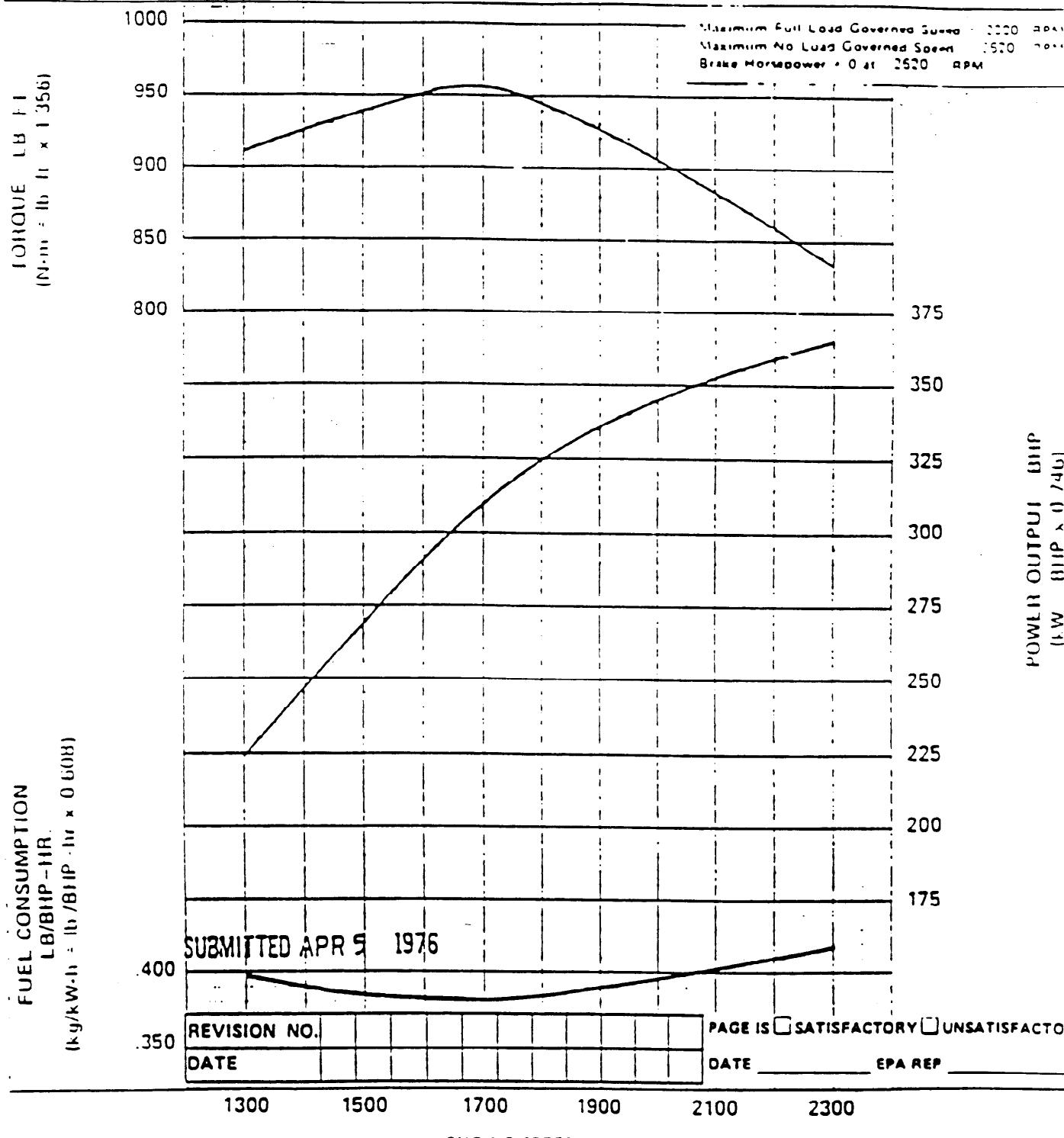
VME

DISPLACEMENT 955 cu in 14.0 liter      ASPIRATION Turbocharged  
 BORE: 5.5 in 140 mm      STROKE: 6.0 in 152 mm      NO OF CYLINDERS 6  
 EMISSION CONTROL: AFC      FUEL SYSTEM: PT

RATING

4P (kW) @ RPM

365 (272) @ 2300



Curves shown above represent engine performance capabilities at SAE standard J816b conditions of 500 ft. (150m) altitude (29.00" Hg (736mm Hg) dry barometer), 65°F (18°C) air intake temperature, and 0.38" Hg (9.6mm Hg) water vapor pressure with No. 2 diesel fuel. The engine may be operated without changing the fuel setting up to 7,000 ft. (2100 m) altitude and 100°F (38°C). For sustained operation at higher altitudes and temperatures the fuel rate of the engines should be adjusted to limit performance by 4% per 1,000 ft (300m) above 7,000 ft. (2100m) and 1% per 10°F above 100°F (2% per 11°C above 38°C).

STANDARDS DEPT.

CERTIFIED WITHIN 5%

CHIEF ENGINEER



CUMMINS ENGINE COMPANY, INC

Columbus, Indiana 47201

Basic Engine Model:  
L10-310Curve Number:  
C-4709

## AUTOMOTIVE PERFORMANCE CURVE

Engine Family:

D34

CPL Code:

1349

Date:

30Apr90

Ex

BMM

Displacement: 611 in.<sup>3</sup> (10.0 litre)

Aspiration: TURBOCHARGED &amp; CHARGE AIR COOLED

Bore: 4.92 in. (125 mm)

Stroke: 5.35 in. (136 mm)

No. of Cylinders: 6

BHP (kW) @ RPM

Emission Control: AFC

Fuel System: PT-STC

Advertised BHP

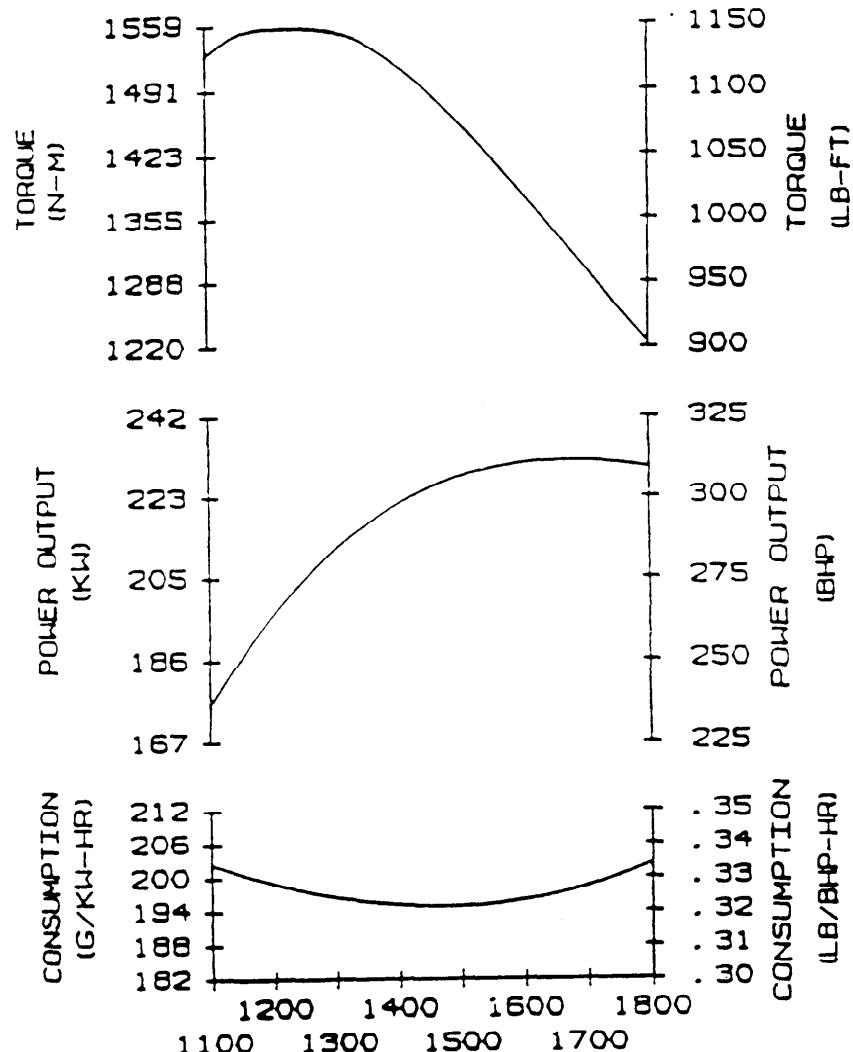
310 (231) @ 1600

Governed Speed

@ 1800

All data is based on the engine operating with fuel system, water pump, lubricating oil pump, air compressor (unloaded), and 10 in H<sub>2</sub>O (250 mm) inlet air restriction and with 2.0 in Hg (50 mm) exhaust restriction; not included are alternator, fan, optional equipment and driven components.

Maximum Full Load Governed Speed = 1800 RPM  
 Maximum No Load Governed Speed = 2106 RPM  
 Brake Horsepower = 0 @ 2106 RPM



Curves shown above represent gross engine performance capabilities obtained and corrected in accordance with SAE J1349 conditions of 29.61 in (100 kPa) barometric pressure (300 ft (90 m) altitude), 77°F (25°C) inlet air temperature, and 0.30 in Hg (1 kPa) water vapor pressure with No. 2 die fuel. The engine may be operated in a transient mode without changing the fuel setting up to 12,000 ft (3,600 m) altitude and 100°F (38°C). For sustained operation at high altitude or temperature the fuel rate of the engine should be adjusted to limit performance by 4% per 1,000 ft (300 m) above 12,000 ft (3,600 m) and 1% per 10°F above 100°F (2% per 11°C above 38°C).

## FURTHER INVESTIGATION

- Option 1: acquire and test a number of production engines making note of the fuel consumption and timing
- Option 2: go directly to the manufacturers and ask them if they have the electronic "cycle beating" strategy

## **CONCLUSIONS**

- **Electronic controls are being used to tailor and engine's performance to the transient test**
- **The fuel consumption is inversely related to the Nox emitted**
- **The transient test is still representative**
  - **Most drivers will drive at maximum power rather than at maximum torque**
  - **The % RPM at which maximum power occurs has not changed significantly**
- **Electronic control may be used to allow the engine to operate "cleaner" during testing than during actual road use**

## **APPENDIX**

### **TABULAR EMISSION AND FUEL CONSUMPTION DATA FOR 1991 CERTIFIED HHDD ENGINES**

Here is a table of the HDV diesel engines certified in 1991 under the heavy service class.

<u>ENGINE MODEL</u>	<u>M/E*</u>	<u>RATED HP</u>	<u>HC</u>	<u>CO</u>	<u>NOX</u>	<u>PARTICULATES</u>
<b>Cummins Engine Company, Inc.</b>						
LTA10	M	300	0.5	W**	5.0	0.37
L10-330E	E	300	0.2	3.8	4.5	0.21
L10-310	M	310	0.6	1.7	4.7	0.19
	M	310	0.6	2.3	4.8	0.25
L10-310E	E	310	0.3	3.1	3.9	0.28
NTC-350	M	350	0.6	W	5.1	0.49
FLEET 300	M	300	0.4	W	5.1	0.40
N14-460E	E	460	0.3	2.1	4.4	0.22
N14-350E	E	350	0.4	2.1	4.9	0.25
N14-410	M	410	0.5	2.0	4.5	0.19
N14-350P	M	350	0.6	2.3	4.3	0.20
NTC-444	M	444	1.0	W	4.7	0.52
N14-330P	M	330	0.4	2.6	5.4	0.20
<b>Caterpillar Inc.</b>						
3176	E	325	0.2	3.5	4.4	0.25
	E	325	0.2	3.2	4.9	0.17
3306B-ATAAC	M	300	0.7	2.3	4.8	0.44
3406B	M	425	0.2	1.3	5.0	0.25
3406B PEEC	M	460	0.4	1.6	4.8	0.23
<b>Detroit Diesel Company</b>						
6L-71TA DDEC ALCC		330	0.4	1.6	4.6	0.29
	E	240	0.5	1.6	4.8	0.33
6L-71TA DDEC COACH		270	0.6	1.8	4.4	0.30
	E	270	0.5	1.5	4.8	0.30
6V-92TA DDEC	E	300	0.4	2.3	4.9	0.25
6V-92TA DDEC COACH		277	0.4	2.6	4.7	0.31
	E	253	0.4	2.1	4.8	0.25
6V-92TA METHANOL COACH		253	0.8	12.0	3.0	0.06
	E	277	0.4	4.8	2.3	0.06
8V-92TA DDEC	E	370	0.5	1.5	4.7	0.20
	E	450	0.4	1.9	4.7	0.25
SERIES 60, 11.1L		275	0.1	2.4	4.9	0.15
	E	350	0.1	2.2	4.7	0.13
SERIES 60, 12.7L		355	0.1	2.3	4.8	0.19
	E	450	0.1	1.8	4.7	0.18
	E	450	0.1	1.7	4.8	0.17

\* Electrical/Mechanical Fuel System

\*\* CO Waiver

<u>ENGINE MODEL</u>	<u>E/M*</u>	<u>RATED HP</u>	<u>HC</u>	<u>CO</u>	<u>Nox</u>	<u>PARTICULATES</u>
---------------------	-------------	-----------------	-----------	-----------	------------	---------------------

**Mack Trucks, Inc.**

EM7-300	M	300	0.1	W**	4.7	0.20
E7-300	M	300	0.1	W	4.6	0.19
EM7-300	E	300	0.1	W	4.4	0.24
E7-400	E	400	0.1	W	4.8	0.16
EM7-275	M	275	0.1	W	4.7	0.20
EM7-300	M	300	0.1	W	4.7	0.20
E7-400	E	400	0.1	W	4.8	0.15
EM7-300	E	300	0.1	W	4.4	0.24
E9-500	M	500	0.5	W	5.3	0.32

**Volvo White Truck Corporation**

TD123EC	E	360	0.3	1.7	4.9	0.21
TD123EB	E	330	0.4	1.6	4.8	0.21
TD123EA	E	300	0.4	1.6	4.9	0.22

\* Electrical/Mechanical Fuel System

\*\* CO Waiver

Note: I assume that the units for the emissions are standard.

Ford Motor Company, General Motors Corporation, Hino Motors, Ltd., Mercedes-Benz Aktiengesellschaft, Navistar International Transportation Corp., Perkins Engine Company, and Renault Vehicles Industrielles have all certified HDV diesel engines in 1991. They are not listed because none of their engines were in the Heavy intended service class.

Company	Engine Model	Rated hp	at RPM	w/Fuel Rate (lb/hr)	Peak Torque (ft-lbf)	at RPM	w/Fuel Rate (LB/HR)	RBSFC ** (lb/BHP-hr)	CBSFC * (lb/BHP-hr)
Cummins Engine Company	LTA10	315	2100	102	1025	1300	82	0.324	
	L10-330E	340	1600	110	1260	1200	93	0.324	0.381
	L10-310	313	1600	104	1169	1200	90	0.332	0.381
	L10-310	316	1600	102	1178	1200	87	0.323	
	L10-310E	306	1600	104	1140	1200	90	0.340	
	NTC-350	367	2100	125	1226	1300	98	0.341	
	FLEET 300	288	1700	100	1091	1100	79	0.347	
	N14-460E	464	1700	157	1564	1200	117	0.338	0.416
	N14-350E	335	1600	112	1332	1200	100	0.334	0.414
	N14-350P	357	1600	119	1386	1100	96	0.333	0.399
Caterpillar Inc.	3176	325	1800	101	1250	1200	94	0.311	0.334
	3406B	400	2000	135	1400	1200	112	0.338	0.361
	3406B PEEC	425	1800	142	1550	1200	118	0.334	0.358
Detroit Diesel Corporation	6L-71TA DDEC (ALCC)	330	2100	121	1050	1200	91	0.367	0.434
	6L-71TA DDEC (ALCC)	270	2100	101	900	1200	76	0.374	
	6V-92TA DDEC	350	2100	135	1020	1200	82	0.386	0.449
	6V-92TA DDEC	300	1800	109	975	1200	78	0.363	
	6V 92TA DDEC COACH	277	2100	105	880	1200	70	0.379	
	6V 92TA DDEC COACH	253	2100	96	775	1200	61	0.379	
	8V-92TA DDEC	450	2100	174	1425	1200	116	0.387	0.437
	8V-92TA DDEC	370	2100	145	1150	1200	113	0.391	
	SERIES 60-11.1L	350	1800	110	1250	1200	94	0.314	0.363
	SERIES 60-11.1L	275	1800	86	1250	1200	94	0.313	
	SERIES 60-12.7L	450	2100	148	1450	1200	106	0.329	0.361
	SERIES 60-12.7L	355	1800	110	1450	1200	106	0.310	

\* Cycle Brake Specific Fuel Consumption

\*\* Rated Brake Specific Fuel Consumption

Company	Engine Model	Rated hp	at RPM	w/Fuel Rate (lb/hr)	Peak Torque (ft-lbf)	at RPM	w/Fuel Rate (lb-hr)	RBSFC ** (lb/BHP-hr)	CBSFC * (lb/BHP-hr)
Mack Trucks Inc.	EM7-300	300	1750	114	1425	1020	95	0.380	0.406
	E7-300	300	1950	124	1083	1200	84	0.413	0.417
	EM7-300	300	1750	102	1425	1020	92	0.340	0.388
	EM7-275	275	1750	105	1305	1020	88	0.382	0.390
	EM7-300	300	1750	114	1425	1020	95	0.380	0.406
	EM7-300	300	1750	102	1425	1020	92	0.340	0.388
	E9-500	500	1900	189	1660	1300	140	0.378	0.431
Volvo White Truck Corporation	TD123EC	356	1900	132	1275	1200	100	0.371	

• Cycle Brake Standard Fuel Consumption

• Rated Brake Standard Fuel Consumption

Note: Caterpillar data taken from 1992 HDV Certification Application  
(engines used are carry-overs)

## APPENDIX 2

### 1991 Cummins HHDD Engine

Peak horsepower at:  $\frac{1800 - 700}{2106 - 700} = 78\% \text{ RPM}$

### 1976 Cummins HHDD Engine

Peak horsepower at:  $\frac{2300 - 800}{2620 - 800} = 82\% \text{ RPM}$

Where:

Peak horsepower  $\frac{\text{Max Power RPM} - \text{Idle RPM}}{\text{Max Rated RPM} - \text{Idle RPM}} = \% \text{ RPM}$   
occurs at:

There is no significant change between the % RPM at which the engine operates. Therefore, the transient test is still representative for HHDD vehicles.